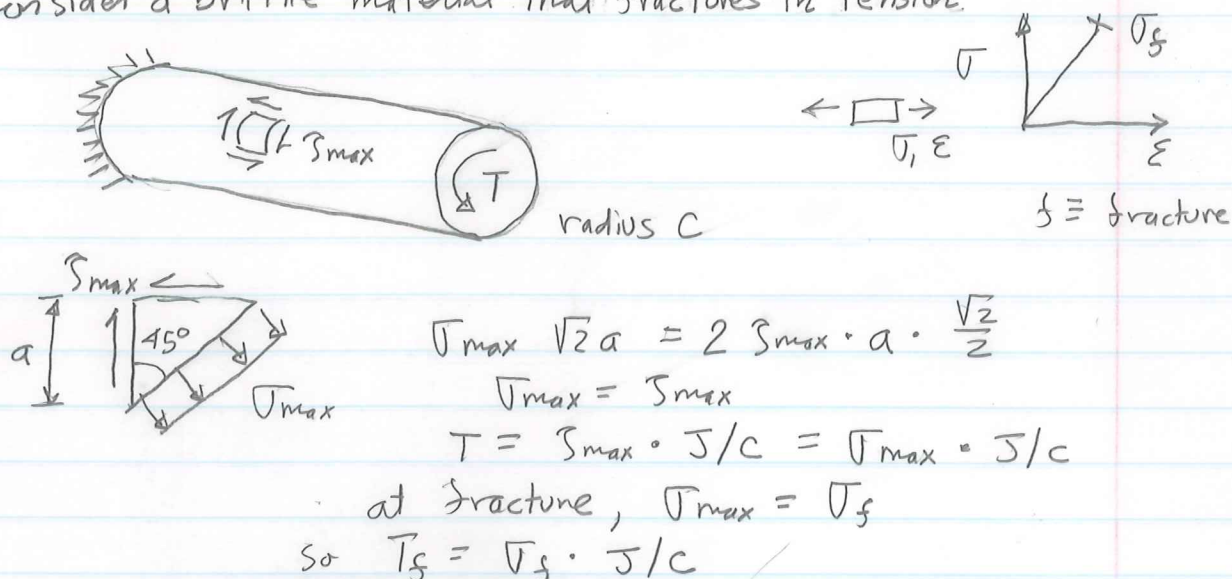
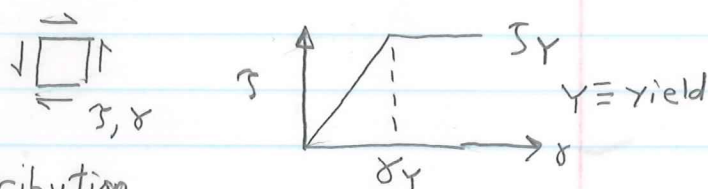


12. Ultimate value of torque T for solid circular shaft (homogeneous, isotropic material)

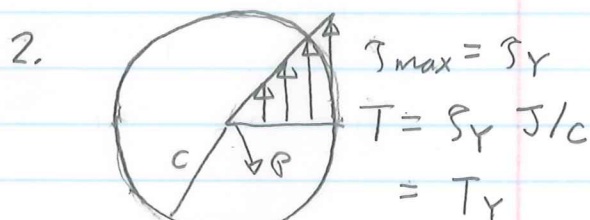
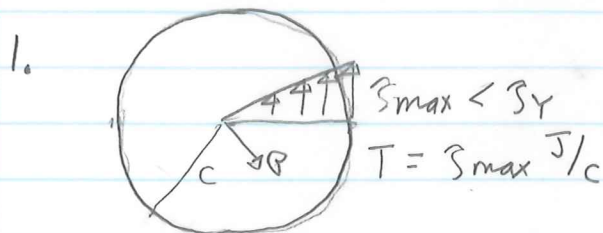
Consider a brittle material that fractures in tension.



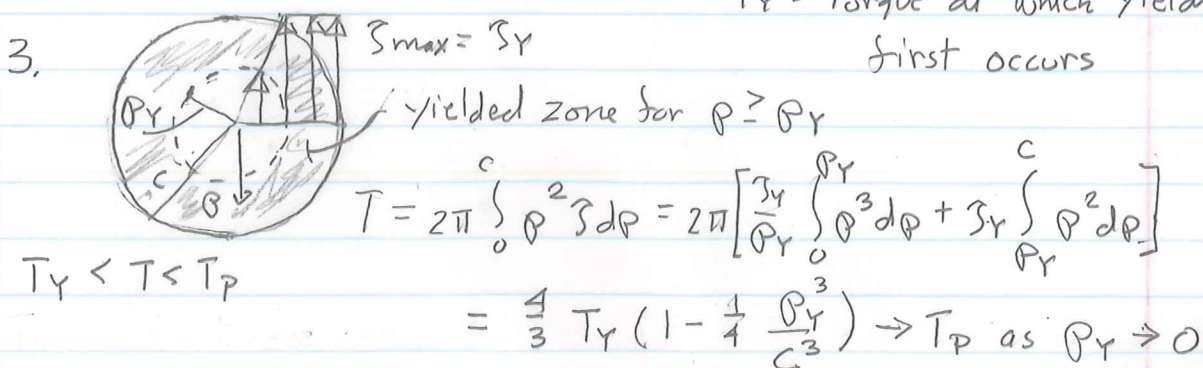
Consider a ductile material with elastic-plastic behavior



Examine the shear stress distribution on a cross-section as T increases (states 1, 2 and 3).



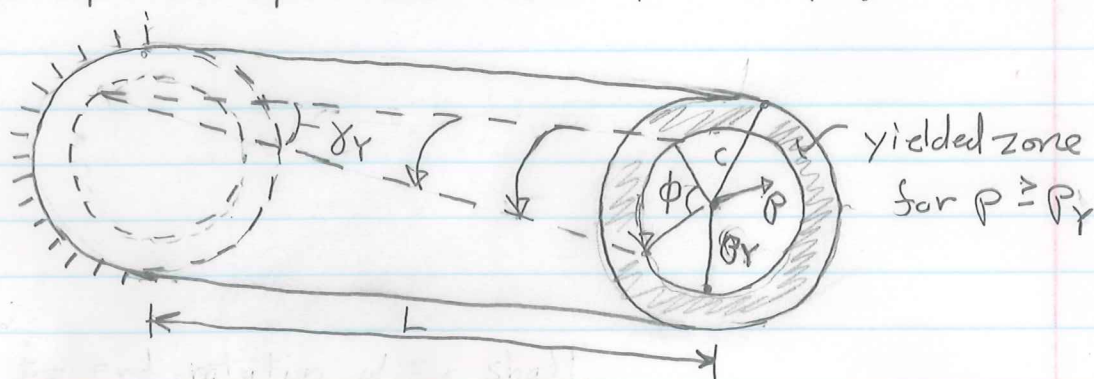
$T_y = \text{Torque at which yielding first occurs}$



where T_p is the Torque when the cross-section is fully yielded (plastic torque).

$$T_p = \frac{4}{3} T_y$$

The shear strain γ is still a linear function of ρ , which means each cross-section still rotates as a rigid planar section. Consider the twisted shaft under a torque T where $T_y < T < T_p$ (state 3) on previous page.

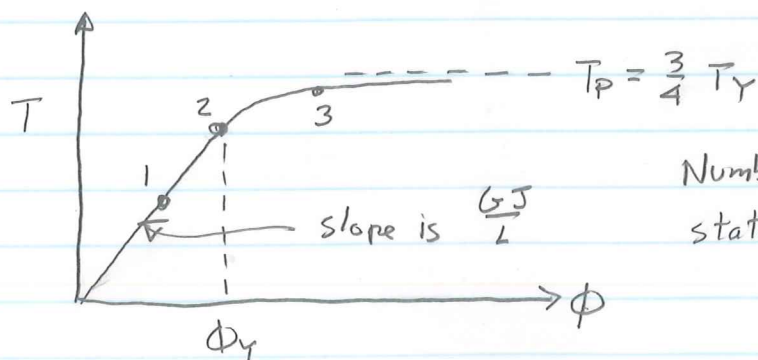


ϕ is the end rotation of the shaft. From geometry,
 $\gamma_y \cdot L = \phi \cdot \rho_y$. Define ϕ_y as the end rotation when yielding first occurs, i.e., when $\rho_y = c$. Then,
 $\gamma_y \cdot L = \phi_y \cdot c$. Thus, $\frac{\rho_y}{c} = \frac{\phi_y}{\phi}$

This expression can be substituted into the equation for T in state 3:

$$T = \frac{4}{3} T_y \left(1 - \frac{1}{4} \frac{\phi_y^3}{\phi^3} \right)$$

T can now be plotted as a function of end rotation ϕ .



Numbers correspond to states on previous page.